

Homework 2 Solutions

Problem 1

$$S_y = 350 \text{ MPa.}$$

$$\text{MSS: } \sigma_1 - \sigma_3 = S_y / n \Rightarrow n = \frac{S_y}{(\sigma_1 - \sigma_3)}$$

$$\text{DE: } \sigma' = (\sigma_A^2 - \sigma_A \sigma_B + \sigma_B^2)^{1/2} = \frac{S_y}{n} \Rightarrow n = \frac{S_y}{\sigma'}$$

$$\text{(a) MSS: } \sigma_1 = 100 \text{ MPa, } \sigma_3 = 0 \Rightarrow n = \frac{350}{100 - 0} = 3.5 \quad \text{Ans.}$$

$$\text{DE: } n = \frac{350}{[100^2 - (100)(100) + 100^2]^{1/2}} = 3.5 \quad \text{Ans.}$$

$$\text{(b) MSS: } \sigma_1 = 100, \sigma_3 = -100 \text{ MPa} \Rightarrow n = \frac{350}{100 - (-100)} = 1.75 \quad \text{Ans.}$$

$$\text{DE: } n = \frac{350}{[100^2 - (100)(-100) + (-100)^2]^{1/2}} = 2.02 \quad \text{Ans.}$$

$$\text{(c) MSS: } \sigma_1 = 0, \sigma_3 = -100 \text{ MPa} \Rightarrow n = \frac{350}{0 - (-100)} = 3.5 \quad \text{Ans.}$$

$$\text{DE: } n = \frac{350}{[(-50)^2 - (-50)(-100) + (-100)^2]^{1/2}} = 4.04 \quad \text{Ans.}$$

Problem 2

$S_{yt} = 60 \text{ kpsi}$, $S_{yc} = 75 \text{ kpsi}$. Eq. (5-26) for yield is

$$n = \left(\frac{\sigma_1}{S_{yt}} - \frac{\sigma_3}{S_{yc}} \right)^{-1}$$

$$\text{(a) } \sigma_1 = 25 \text{ kpsi, } \sigma_3 = 0 \Rightarrow n = \left(\frac{25}{60} - \frac{0}{75} \right)^{-1} = 2.40 \quad \text{Ans.}$$

$$\text{(b) } \sigma_A, \sigma_B = \frac{-12 + 15}{2} \pm \sqrt{\left(\frac{-12 - 15}{2} \right)^2 + (-9)^2} = 17.7, -14.7 \text{ kpsi}$$

$$\sigma_1 = 17.7, \sigma_2 = 0, \sigma_3 = -14.7 \text{ kpsi} \Rightarrow n = \left(\frac{17.7}{60} - \frac{-14.7}{75} \right)^{-1} = 2.04 \quad \text{Ans.}$$

Problem 3

$S_{ut} = 30 \text{ kpsi}$, $S_{uc} = 90 \text{ kpsi}$

BCM: Eqs. (5-31), p. 250

MM: see Eqs. (5-32), p. 250

$$(a) \sigma_A, \sigma_B = \frac{-15+10}{2} \pm \sqrt{\left(\frac{-15-10}{2}\right)^2 + (-15)^2} = 17.03, -22.03 \text{ kpsi}$$

$$\text{BCM: Eq. (5-31b), } n = \left(\frac{17.03}{30} - \frac{-22.03}{90}\right)^{-1} = 1.23 \text{ Ans.}$$

MM: $\sigma_A \geq 0 \geq \sigma_B$, and $|\sigma_B/\sigma_A| \geq 1$, Eq. (5-32b),

$$n = \left[\frac{(S_{uc} - S_{ut})\sigma_A}{S_{uc}S_{ut}} - \frac{\sigma_B}{S_{uc}}\right]^{-1} = \left[\frac{(90-30)17.03}{90(30)} - \frac{-22.03}{90}\right]^{-1} = 1.60 \text{ Ans.}$$

(b) $\sigma_A = 15$ kpsi, $\sigma_B = -15$ kpsi,

$$\text{BCM: Eq. (5-31a), } n = \left(\frac{15}{30} - \frac{-15}{90}\right)^{-1} = 1.5 \text{ Ans.}$$

MM: $\sigma_A \geq 0 \geq \sigma_B$, and $|\sigma_B/\sigma_A| \leq 1$, Eq. (5-32a), $n = \frac{S_{ut}}{\sigma_A} = \frac{30}{15} = 2.0 \text{ Ans.}$

Problem 4

From Table A-20, $S_y = 370$ MPa. From the solution of Prob. 3-69, in the plane of analysis

$$\sigma_1 = 275 \text{ MPa, } \sigma_2 = -12.1 \text{ MPa, and } \tau_{\max} = 144 \text{ MPa}$$

The state of stress is *plane stress*. Thus, the three-dimensional principal stresses are

$$\sigma_1 = 275 \text{ MPa, } \sigma_2 = 0, \text{ and } \sigma_3 = -12.1 \text{ MPa}$$

$$\text{MSS: From Eq. (5-3), } n = \frac{S_y}{\sigma_1 - \sigma_3} = \frac{370}{275 - (-12.1)} = 1.29 \text{ Ans.}$$

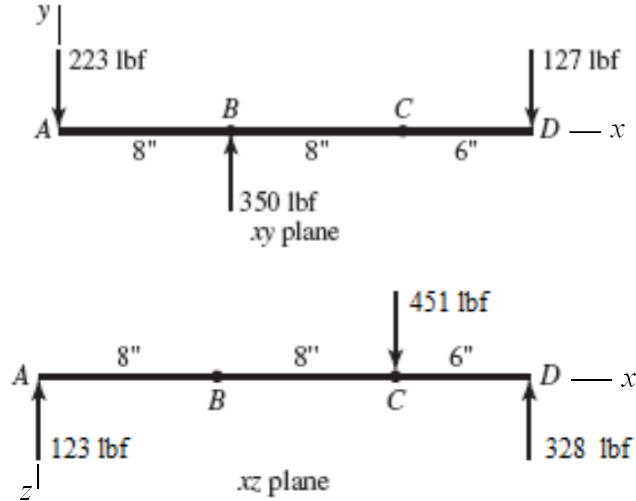
DE: From Eqs. (5-13) and (5-19)

$$n = \frac{S_y}{\sigma'} = \frac{S_y}{(\sigma_A^2 - \sigma_A\sigma_B + \sigma_B^2)^{1/2}} = \frac{370}{\left[275^2 - 275(-12.1) + (-12.1)^2\right]^{1/2}} = 1.32 \text{ Ans.}$$

Problem 5

From Table A-20, for AISI 1035 CD, $S_y = 67$ kpsi.

From force and bending-moment equations, the ground reaction forces are found in two planes as shown.



The maximum bending moment will be at B or C . Check which is larger. In the xy plane,

$$M_B = 223(8) = 1784 \text{ lbf} \cdot \text{in} \text{ and } M_C = 127(6) = 762 \text{ lbf} \cdot \text{in}.$$

In the xz plane, $M_B = 123(8) = 984 \text{ lbf} \cdot \text{in}$ and $M_C = 328(6) = 1968 \text{ lbf} \cdot \text{in}$.

$$M_B = [(1784)^2 + (984)^2]^{1/2} = 2037 \text{ lbf} \cdot \text{in}$$

$$M_C = [(762)^2 + (1968)^2]^{1/2} = 2110 \text{ lbf} \cdot \text{in}$$

So point C governs. The torque transmitted between B and C is $T = (300 - 50)(4) = 1000 \text{ lbf} \cdot \text{in}$. The stresses are

$$\tau_{xz} = \frac{16T}{\pi d^3} = \frac{16(1000)}{\pi d^3} = \frac{5093}{d^3} \text{ psi}$$

$$\sigma_x = \frac{32M_C}{\pi d^3} = \frac{32(2110)}{\pi d^3} = \frac{21492}{d^3} \text{ psi}$$

For combined bending and torsion, the maximum shear stress is found from

$$\tau_{\max} = \left[\left(\frac{\sigma_x}{2} \right)^2 + \tau_{xz}^2 \right]^{1/2} = \left[\left(\frac{21.49}{2d^3} \right)^2 + \left(\frac{5.09}{d^3} \right)^2 \right]^{1/2} = \frac{11.89}{d^3} \text{ kpsi}$$

Max Shear Stress theory is chosen as a conservative failure theory. From Eq. (5-3)

$$\tau_{\max} = \frac{S_y}{2n} = \frac{11.89}{d^3} = \frac{67}{2(2)} \quad \Rightarrow \quad d = 0.892 \text{ in} \quad \text{Ans.}$$